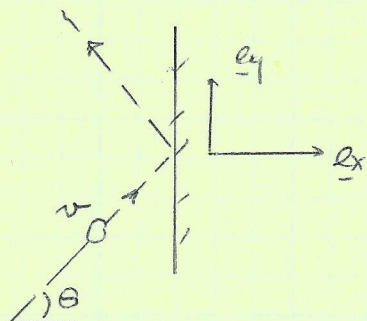


1.



$$\underline{t} = \pm \underline{e}_y,$$

$$\underline{n} = \pm \underline{e}_x.$$

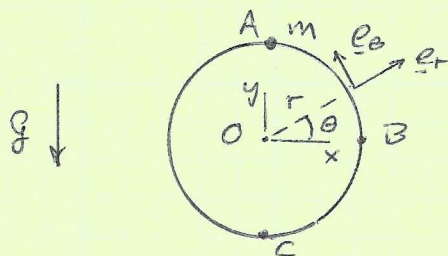
Just before impact, $\underline{v} = v(\cos\theta \underline{e}_x + \sin\theta \underline{e}_y)$.

Just after impact (from eq. (*) on p. 97 of Notes),

$$\begin{aligned} \underline{v}' &= (\underline{t} \cdot \underline{v}) \underline{t} - e (\underline{n} \cdot \underline{v}) \underline{n} \\ &= (\underline{e}_y \cdot \underline{v}) \underline{e}_y - e (\underline{e}_x \cdot \underline{v}) \underline{e}_x \end{aligned}$$

$$\Rightarrow \underline{v}' = v \sin\theta \underline{e}_y - e v \cos\theta \underline{e}_x.$$

2.



smooth ring -
no friction.

\Rightarrow conservative problem.

$$E = K + U, \text{ where } K = \frac{1}{2}mv^2 \text{ and } U = mgy (+\text{const.})$$

$$E_A = E_B \Rightarrow \frac{1}{2}mv_B^2 = mgr \Rightarrow \underline{v_B = \sqrt{2gr}}.$$

$$E_A = E_C \Rightarrow \frac{1}{2}mv_C^2 - mgr = mgr \Rightarrow \underline{v_C = \sqrt{4gr} = 2\sqrt{gr}}.$$

$$(\text{check: } E_B = E_C \Rightarrow \frac{1}{2}mv_C^2 - mgr = \frac{1}{2}mv_B^2 \Rightarrow v_C^2 = v_B^2 + 2gr = 4gr \checkmark).$$

$$\begin{aligned} \underline{H}_O &= \underline{r} \times m \underline{v} = r \underline{e}_r \times m v \underline{e}_t \\ &= r m v \underline{e}_r \times (-\underline{e}_\theta) = -r m v \underline{e}_z. \end{aligned}$$

$$\Rightarrow |\underline{H}_O| = r m v.$$

$$(a) \quad |\underline{H}_O|_B = r m v_B = \underline{r m \sqrt{2gr}}.$$

$$(b) \quad |\underline{H}_O|_C = r m v_C = \underline{r m \sqrt{4gr}}.$$

2, continued:

$$(c) \quad \dot{H}_0 = (\underline{r} \times m \underline{v})' = \underbrace{\underline{v} \times m \underline{v}}_0 + \underline{r} \times m \dot{\underline{v}} \\ = \underline{r} \times m \underline{a} = \underline{r} \times \underline{F},$$

where $\underline{F} = N \underline{e}_r - mg \underline{e}_y$ (N = normal force by track on mass - no friction).

Thus,

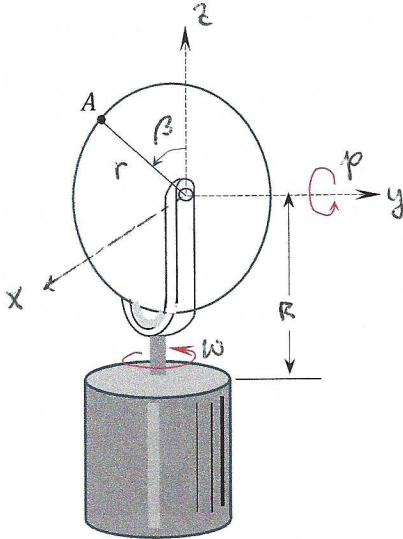
$$\begin{aligned} \dot{H}_0 &= r \underline{e}_r \times (N \underline{e}_r - mg \underline{e}_y) \\ &= -mgr \underline{e}_r \times \underline{e}_y \\ &= -mgr (\cos \theta \underline{e}_x + \sin \theta \underline{e}_y) \times \underline{e}_y \\ &= -mgr \cos \theta \underline{k}. \end{aligned}$$

at point B, $\theta = 0 \Rightarrow \underline{\dot{H}_0} = -mgr \underline{k}$

at point C, $\theta = -\frac{\pi}{2} \Rightarrow \underline{\dot{H}_0} = \underline{0}$.

3.

A circular disk of radius r is mounted in a bearing in a yoke and spins with constant angular velocity $\dot{\beta} = p$. The yoke also revolves with constant angular velocity ω counterclockwise as shown. Find \mathbf{a}_A in terms of the angle β .



Solution

The inertial reference system¹ is fixed to the ground. The rotating system axes are labeled in the figure as xyz . They are attached to the yoke at the center of the spinning disk and rotate at angular velocity $\omega \hat{\mathbf{k}}$. Note that the basis vectors associated with the xyz system are denoted as $(\hat{\mathbf{i}}, \hat{\mathbf{j}}, \hat{\mathbf{k}})$.

Point A on the disk moves so that it has a velocity relative to the rotating system.

$$\mathbf{r}_{AB} = r \sin \beta \hat{\mathbf{i}} + r \cos \beta \hat{\mathbf{k}}$$

$$\mathbf{v}_{rel} = p \hat{\mathbf{j}} \times \mathbf{r}_{AB} = \begin{vmatrix} \hat{\mathbf{i}} & \hat{\mathbf{j}} & \hat{\mathbf{k}} \\ 0 & p & 0 \\ r \sin \beta & 0 & r \cos \beta \end{vmatrix} = rp \cos \beta \hat{\mathbf{i}} - rp \sin \beta \hat{\mathbf{k}}$$

$$\mathbf{a}_A = \ddot{\omega} \times \mathbf{r}_{AB} + \omega \times \omega \times \mathbf{r}_{AB} + 2\omega \times \mathbf{v}_{rel} + \mathbf{a}_{rel}$$

$$\omega \times \mathbf{r}_{AB} = \begin{vmatrix} \hat{\mathbf{i}} & \hat{\mathbf{j}} & \hat{\mathbf{k}} \\ 0 & 0 & \omega \\ r \sin \beta & 0 & r \cos \beta \end{vmatrix} = \omega r \sin \beta \hat{\mathbf{j}}$$

$$\omega \times (\omega \times \mathbf{r}_{AB}) = \begin{vmatrix} \hat{\mathbf{i}} & \hat{\mathbf{j}} & \hat{\mathbf{k}} \\ 0 & 0 & \omega \\ 0 & \omega r \sin \beta & 0 \end{vmatrix} = -\omega^2 r \sin \beta \hat{\mathbf{i}}$$

$$2\omega \times \mathbf{v}_{rel} = \begin{vmatrix} \hat{\mathbf{i}} & \hat{\mathbf{j}} & \hat{\mathbf{k}} \\ 0 & 0 & 2\omega \\ rp \cos \beta & 0 & -rp \sin \beta \end{vmatrix} = 2\omega rp \cos \beta \hat{\mathbf{j}}$$

$$\begin{aligned} \mathbf{a}_{rel} &= p \hat{\mathbf{j}} \times (p \hat{\mathbf{j}} \times \mathbf{r}_{AB}) = p \hat{\mathbf{j}} \times [p \hat{\mathbf{j}} \times (r \sin \beta \hat{\mathbf{i}} + r \cos \beta \hat{\mathbf{k}})] = p \hat{\mathbf{j}} \times (pr \cos \beta \hat{\mathbf{i}} - pr \sin \beta \hat{\mathbf{k}}) \\ &= -p^2 r \cos \beta \hat{\mathbf{k}} - p^2 r \sin \beta \hat{\mathbf{i}} \end{aligned}$$

Note that \mathbf{a}_{rel} is the acceleration of point A relative to the axes xyz that rotate at constant angular velocity $p \hat{\mathbf{j}}$.

$$\mathbf{a}_A = -\omega^2 r \sin \beta \hat{\mathbf{i}} + 2\omega rp \cos \beta \hat{\mathbf{j}} - p^2 r \cos \beta \hat{\mathbf{k}} - p^2 r \sin \beta \hat{\mathbf{i}}$$

$$\mathbf{a}_A = -(p^2 + \omega^2) r \sin \beta \hat{\mathbf{i}} + 2\omega rp \cos \beta \hat{\mathbf{j}} - p^2 r \cos \beta \hat{\mathbf{k}}$$

¹ An inertia reference frame is one that is not accelerating.