

University of California, Berkeley
 Department of Mechanical Engineering
 ME 104, Spring 2021

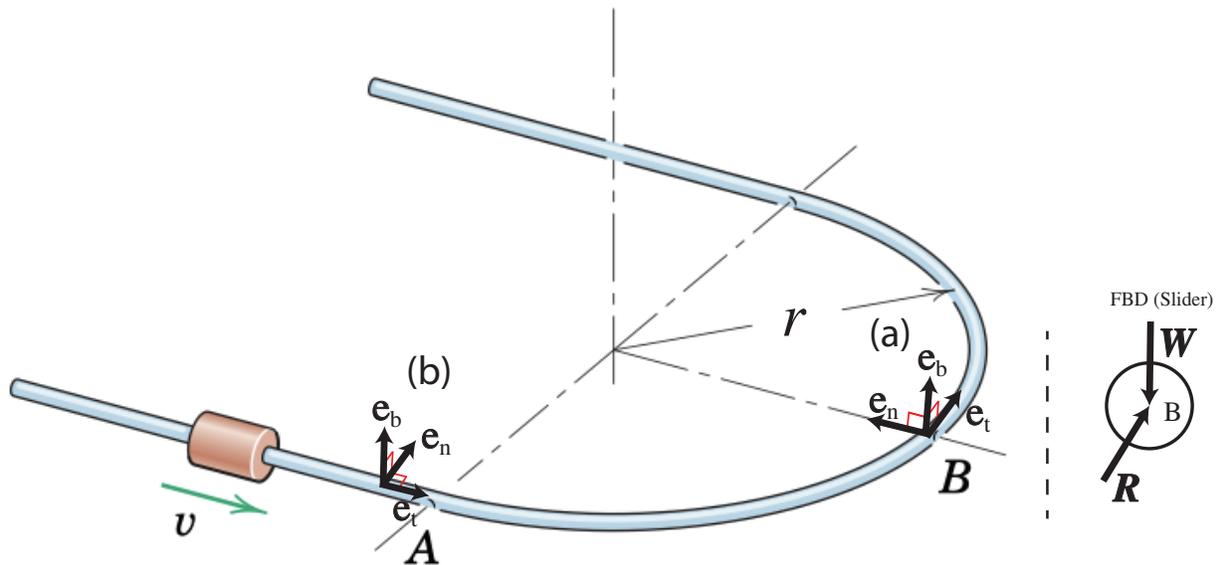
Midterm Exam 1 (15 April 2021)

SOLUTIONS

Problem 1 25 points

(a) (5 pts)

- Frenet-Serret basis and free-body diagram of the slider at B



- Applying Frenet-Serret basis, reaction force (\mathbf{R}) supplied by the rod can be written as

$$\mathbf{R} = R_t \mathbf{e}_t + R_n \mathbf{e}_n + R_b \mathbf{e}_b,$$

where $\mathbf{W} = mg(-\mathbf{e}_b)$ is the weight of the slider.

(b) (4 pts)

The basis for the straight-line portion is given in the figure above. Note that \mathbf{e}_n is defined such that the basis joints continuously with the Frenet-Serret basis for the semi-circular portion.

(c) (16 pts)

- Passing point A :
 Applying Newton's second law

$$\begin{aligned} \mathbf{F} &= m\mathbf{a} \\ \Rightarrow R_t \mathbf{e}_t + R_n \mathbf{e}_n + R_b \mathbf{e}_b - mg \mathbf{e}_b &= m(\dot{v} \mathbf{e}_t + \kappa v^2 \mathbf{e}_n) = m\dot{v} \mathbf{e}_t. \end{aligned}$$

Taking dot products,

$$\begin{aligned} (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_t &\Rightarrow R_t = m\dot{v} = 0 \\ (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_n &\Rightarrow R_n = 0 \\ (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_b &\Rightarrow R_b = mg \end{aligned}$$

$$\therefore \|\mathbf{R}\| = mg \quad \text{N}$$

- Passing point B :

Applying Newton's second law

$$\begin{aligned} \mathbf{F} &= m\mathbf{a} \\ \Rightarrow R_t \mathbf{e}_t + R_n \mathbf{e}_n + R_b \mathbf{e}_b - mg \mathbf{e}_b &= m(\dot{v} \mathbf{e}_t + \kappa v^2 \mathbf{e}_n), \end{aligned}$$

where $\kappa = \frac{1}{r}$. Taking dot products,

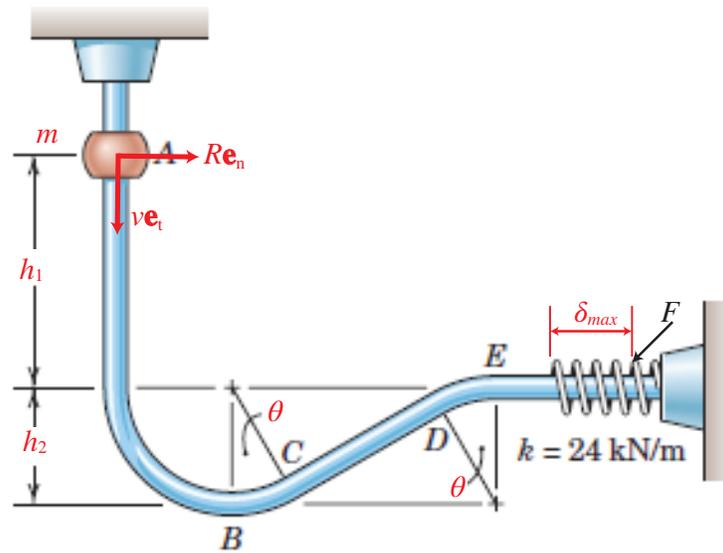
$$\begin{aligned} (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_t &\Rightarrow R_t = m\dot{v} = 0 \\ (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_n &\Rightarrow R_n = \frac{v^2}{r} \\ (\mathbf{F} = m\mathbf{a}) \cdot \mathbf{e}_b &\Rightarrow R_b = mg \end{aligned}$$

$$\therefore \|\mathbf{R}\| = \sqrt{\left(\frac{v^2}{r}\right)^2 + (mg)^2} \quad \text{N}$$

(d)

The existence of jump discontinuity in the normal force at point A can be problematic, which causes unnecessary impact to both track and train.

Problem 2 25 points



(a) (4 pts)

The work done by the reaction force is,

$$W_{12} = \int_{t_1}^{t_2} \mathbf{R} \cdot \mathbf{v} dt = \int_{t_1}^{t_2} R v e_n \cdot e_t dt = \int_{t_1}^{t_2} 0 dt = 0.$$

Therefore, the reaction force is workless and the energy is conserved.

(b) (7 pts)

From the conservation of total mechanical energy,

$$\begin{aligned} T_A + V_A &= T_B + V_B \\ \Rightarrow 0 + mg(h_1 + h_2) &= \frac{1}{2} m v_B^2 + 0 \\ \Rightarrow v_B &= \sqrt{2g(h_1 + h_2)} \quad \text{m/s} \end{aligned}$$

(c) (8 pts)

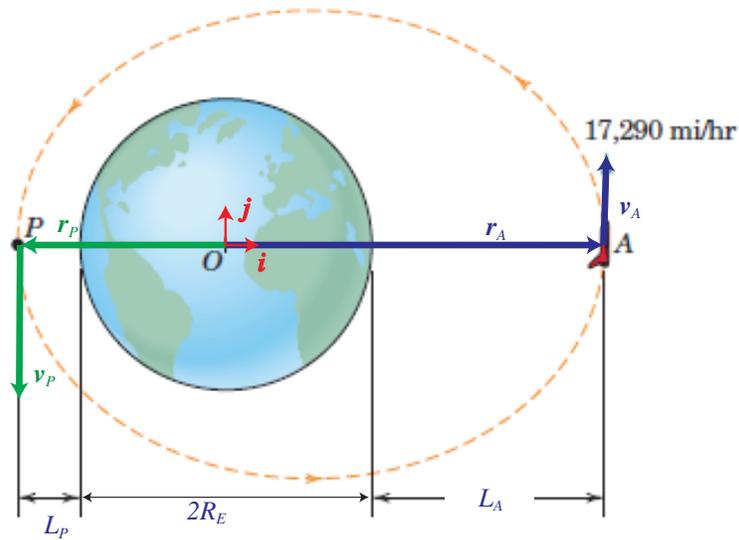
From the similar procedure of (b),

$$\begin{aligned} T_A + V_A &= T_F + V_F \\ \Rightarrow 0 + mg(h_1 + h_2) &= 0 + mgh_2 + \frac{1}{2} k \delta_{max}^2 \\ \Rightarrow \delta_{max} &= \sqrt{\frac{2mgh_1}{k}} \quad \text{m} \end{aligned}$$

(d) (6 pts)

Since the spring deforms without hitting the wall, the motions between the point A and the point F would continue periodically.

Problem 3 20 points



(a) (10pts)

From the conservation of angular momentum,

$$\begin{aligned} \frac{\mathbf{H}^O}{m} &= \mathbf{r}_P \times \mathbf{v}_P = \mathbf{r}_A \times \mathbf{v}_A \\ &\Rightarrow (R_E + L_A)\mathbf{i} \times v_A\mathbf{j} = (R_E + L_B)(-\mathbf{i}) \times v_P(-\mathbf{j}) \\ &\Rightarrow (R_E + L_A)v_A\mathbf{k} = (R_E + L_B)v_P\mathbf{k} \end{aligned}$$

$$\therefore \frac{\mathbf{H}^O}{m} \cdot \mathbf{k} \Rightarrow v_P = \frac{(R_E + L_A)v_A}{R_E + L_B} \quad \text{ft/s or mi/hr}$$

(b) (10pts)

Since the mechanical energy per unit mass is expected to be conserved, we expect to observe

$$\frac{E_P}{m} = \frac{1}{2}v_P^2 - \frac{Gm_e}{r_P} = \frac{E_A}{m} = \frac{1}{2}v_A^2 - \frac{Gm_e}{r_A} \quad \frac{\text{lbf} \cdot \text{ft}}{\text{slug}} \quad \text{or} \quad \text{ft}^2/\text{s}^2$$