Problem 1 (12+15+8 points)

Consider a three-pinned arch of the figure below.



- (a) Draw the free-body diagram of the three-pinned arch, as well as the free body diagrams of the subbodies AB and BC.
- (b) Find the external reactions at points A and C.
- (c) Find the horizontal and vertical forces transmitted through the pin at B.



b)

$$FBD1 : f) \in H_{c} = 0 \implies 2A_{x} - 14A_{y} - 4(3) + 4(3) - 24 = 0$$

$$2A_{x} - 14A_{y} = 24 \implies A_{x} - 7A_{y} = 12 \quad (*)$$

$$FBD2 : f) \in H_{B} = 0 \implies -6A_{y} + 6A_{x} = 0 \implies A_{x} = A_{y} \quad (**)$$
From (*) $t \quad (**) \implies A_{x} = A_{y} = -2 \times N$

FBDI:
$$\Sigma F_y = 0 = 3$$
 Ay - 3 + Cy = 0
=> $C_y = 5 \neq N$

$$\mathcal{E} F_{x} = 0 \implies A_{x} + 3 + C_{x} = 0$$

 $C_{x} = -1 K N$

C)
$$F_{BD2}$$
: $\mathcal{E}F_{y=0} = \mathcal{B}g + Ay=0 = \mathcal{B}g = \mathcal{A}KN$
 $\mathcal{E}F_{x=0} = \mathcal{A}x + F + B_{x} = 0 = \mathcal{B}\frac{B_{x} = -1KN}{B_{x} = -1KN}$

In summary ;

$$A_{x} = -2 \underbrace{i}_{kN} (kN)$$

$$A_{y} = -2 \underbrace{j}_{kN} (kN)$$

$$C_{x} = -1 \underbrace{i}_{kN} (kN)$$

$$C_{y} = 5 \underbrace{j}_{kN} (kN)$$
Pin:
$$I_{KN} \underbrace{j}_{kN} f_{kN} (kN)$$

$$Z_{kN}$$

Problem 2 (4+6+8+12 points)

A system of four forces \mathbf{F}_1 , \mathbf{F}_2 , \mathbf{F}_3 , \mathbf{F}_4 is acting on the orthogonal parallelepiped of the figure below. Let the magnitudes of these forces be $F_1 = 5$ kN, $F_2 = 5\sqrt{2}$ kN, $F_3 = 8$ kN, and $F_4 = 5$ kN, respectively.



- (a) Express each of the four forces in vector form.
- (b) Find the resultant **R** of the four forces.
- (c) Find the moment of \mathbf{F}_2 about the x-axis.
- (d) What should be the relation between the lengths a and b so that the system of four forces be statically equivalent to a force that passes through point A?

(a)
$$F_{1} = -5jkN$$
 (b)
 $F_{2} = 5f_{2}(-\cos 45^{\circ}i + \sin 45^{\circ}j)kN = (-5i+5j)kN0$
 $F_{3} = -8kkN$ (b)
 $F_{4} = 5ikN$ (c)
 $F_{4} = 5ikN$ (c)

(b) $R = F_1 + F_2 + F_3 + F_4 = -8k kN^0$ (c) $M_{y} = (bk \times F_2 \cdot i)i = (bk \times (-5i + 5j) \cdot i)i = (-5bj - 5bj) \cdot i)i = -5bj$

(A)
We require that the resultant moment of all forces
about A must be zero.

$$\therefore \underline{0} = \underline{M}_{A} = \underline{v}_{A} \times \underline{F}_{i} + \underline{v}_{A} \in \underline{F}_{L} + \underline{v}_{A} \otimes \underline{F}_{H} + \underline{v}_{A} \otimes \underline{x} \underline{F}_{H} = (\underline{a} \pm -\underline{a}_{i} + \underline{b} \underline{E}) \times (-5\underline{i}_{i}) + \underline{b} \underline{E} \times (-5\underline{i}_{i} + 5\underline{i}_{i}) + \underline{a} \underline{i} \times (-8\underline{E}) + (\underline{a} \pm -\underline{a}_{i}) \times 5\underline{b}$$

$$= -5\underline{a} \underline{k} + 5\underline{b} \underline{i} + 5\underline{b} \underline{j} - 5\underline{b} \underline{i} + 8\underline{a} \underline{j} + 5\underline{a} \underline{k}$$

$$= (8\underline{a} - 5\underline{b})\underline{\dot{a}}$$

$$\therefore 8\underline{a} - 5\underline{b} = 0$$

$$\therefore \underline{a} = -5\underline{a} \otimes \underline{a}$$

Problem 3 (5+5+5+20 points)

In the hexagonal truss shown in the figure below all truss bars have length 2 m.



- (a) Draw the free-body diagram of the truss.
- (b) Argue that the truss is statically determinate.
- (c) Determine the external reactions at points A and B.
- (d) Determine the forces of members DE, FG, BD and BE and state explicitly if they are in tension or compression.



b) 2j = n+r? 14 = 11+3Also, we have 3 reactions \$ 3 equilibra equations.

(c)
$$\mathcal{E}H_{A}=0 \implies \mathcal{2}B_{y} - I(2) - 2\sqrt{3} = 0$$

 $\implies B_{y} = \mathcal{2}, +3KN + 1$
 $\mathcal{E}F_{y}=\infty \implies A_{y} = B_{y} - I = 1, +3KN + 1$
 $\mathcal{E}F_{x}=0 \implies A_{x} = -2KN = 2KN = 0$
(d) $T_{0R} = \int_{-\infty}^{\infty} T_{0R} = T_{0R} =$

$$\begin{split} \mathcal{E}F_{y} = 0 &= > -T_{0B} \frac{f_{3}}{2} - T_{BE} \frac{f_{3}}{2} + 2.73 = 0 \\ T_{BD} \sqrt{3} &= 2.73 \Rightarrow \boxed{T_{BO} = 1.576 \, \text{kN} \, (\text{c})} \\ \hline[T_{DE} = 1.576 \, \text{kN} \, (\text{c})] \\ \hline[T_{FG} = 1 \, \text{KN} \, (\text{c})] \\ \hline[T_{FG} = 1 \, \text{KN} \, (\text{c})] \\ \end{split}$$

$$EM_{G=0} = > -T_{DE}V_{3} + T_{DR}V_{3} = 0$$

 $T = T_{DE}T_{F} = 1.57 \text{ KN}(T)$