

\* -5 AT LEAST FOR ANY MATHEMATIC ERRORS IN WORK

\* NO CREDIT FOR DIMENSIONALLY INCORRECT WORK

UNIVERSITY OF CALIFORNIA, BERKELEY  
MECHANICAL ENGINEERING  
ME106 Fluid Mechanics  
1st Test, S08 Prof S. Morris

NAME SOLUTIONS

1.(65) (a) Find the fluid acceleration  $\underline{a}$  for plane stagnation point flow in which the velocity is given by the expression  $\underline{V} = cx\mathbf{i} - cy\mathbf{j}$  (the constant  $c > 0$ ).

EITHER 
$$\frac{d\underline{V}}{dt} = \frac{d}{dt}(cx\mathbf{i}) - \frac{d}{dt}(cy\mathbf{j})$$
  

$$= c\mathbf{i} \frac{dx}{dt} - c\mathbf{j} \frac{dy}{dt}$$

Exam stats  
Mean: 124.8/200  
St. Dev: 37.1

( $c_x, c_y$  const in mag. & direction)

But  $\frac{dx}{dt} = cx, \frac{dy}{dt} = -cy \Rightarrow$

$$\frac{d\underline{V}}{dt} = c^2(\mathbf{i}x + \mathbf{j}y)$$

OR.  $\underline{V} = u\mathbf{i} + v\mathbf{j}$

$$\frac{d\underline{V}}{dt} = u \frac{\partial \underline{V}}{\partial x} + v \frac{\partial \underline{V}}{\partial y} \quad \because \frac{\partial \underline{V}}{\partial t} = 0$$

$$\frac{\partial \underline{V}}{\partial x} = c\mathbf{i}, \frac{\partial \underline{V}}{\partial y} = -c\mathbf{j} \Rightarrow$$

$$\frac{d\underline{V}}{dt} = (cx)(c\mathbf{i}) + (-cy)(-c\mathbf{j})$$
  

$$\Rightarrow \underline{a} = c^2(\mathbf{i}x + \mathbf{j}y)$$

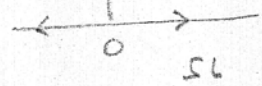
lost points for:  
math errors, -5  
did not underline unit vectors, -5  
confuse vectors and scalars, -10

Pr. 1 Stats  
Mean: 57.4/65  
St Dev: 12.1

(b) By considering the stagnation streamline, explain why  $\underline{a} \neq 0$  in this flow.

on the branch  $x=0$   $\underline{V} = -cy\mathbf{j}$

Velocity of FP moving along SL to origin decreases with  $y \Rightarrow \underline{a} \neq 0$ .



No credit for arguments that:

- velocity changes throughout flow
- using  $\underline{a}$  w/o physical reasoning

IN BLOCK LETTERS PRINT YOUR NAME ON THIS PAGE

Pr. 2 Stats  
 Mean: 40.6/65  
 St. Dev: 22, 1

2. (65) The vector field  $\mathbf{V} = -\frac{1}{2}cr\hat{r} + v(r,t)\hat{\theta} + cz\hat{k}$  ( $c > 0$ , constant) represents a vortex that is being stretched in the axial direction. For this flow, the fluid acceleration is given by the expression

$$\mathbf{a} = \left(\frac{1}{4}c^2r - \frac{v^2}{r}\right)\hat{r} + \left(\frac{\partial v}{\partial t} - \frac{1}{2}c\frac{\partial(rv)}{\partial r}\right)\hat{\theta} + c^2z\hat{k}$$

(You are not required to show that.) Find the partial differential equation satisfied by the function  $v(r,t)$ . (You are not asked to solve it.)

Given. In cylindrical polar coordinates  $r, \theta, z$ , if  $\mathbf{F} = f_r\hat{r} + f_\theta\hat{\theta} + f_z\hat{k}$  is an arbitrary vector, then

$$\nabla \times \mathbf{F} = \frac{1}{r} \begin{vmatrix} \hat{r} & r\hat{\theta} & \hat{k} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ f_r & rf_\theta & f_z \end{vmatrix}$$

Euler's equation for constant  $p$  is satisfied  $\Leftrightarrow \boxed{\nabla \times \underline{a} = 0}$  +20

$$r \nabla \times \underline{a} = \begin{vmatrix} \hat{r} & r\hat{\theta} & \hat{k} \\ \frac{\partial}{\partial r} & 0 & 0 \\ \left\{ \frac{c^2r}{4} - \frac{v^2}{r}, r \left[ \frac{\partial v}{\partial t} - \frac{1}{2}c\frac{\partial(rv)}{\partial r} \right], c^2z \right\} & & \end{vmatrix} = 0\hat{r} - r\hat{\theta} \frac{\partial}{\partial r} \left( \frac{\partial v}{\partial t} - \frac{1}{2}c\frac{\partial(rv)}{\partial r} \right) + 0\hat{k}$$

← +30 ——— ↑  
Expanding  $\nabla \times \underline{a} = 0$

so  $\nabla \times \underline{a} = 0 \Leftrightarrow \frac{\partial}{\partial r} \left( \frac{\partial v}{\partial t} - \frac{1}{2}c\frac{\partial(rv)}{\partial r} \right) = 0$  A.

i.e.  $\boxed{\frac{\partial}{\partial t}(rv) - \frac{1}{2}cr\frac{\partial}{\partial r}(rv) = f(t)}$  B.

Either A or B is acceptable. +15 for complete and correct solution.

\* if you attempted to use Euler's equation and plugged in CORRECTLY, +30

- Point deductions:
- Did not see  $\frac{\partial v}{\partial \theta} + \frac{\partial v}{\partial z} = 0$  -15
  - Missing  $r$  terms, -5

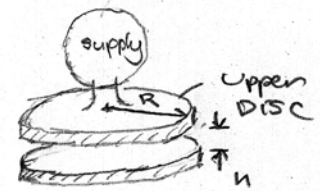
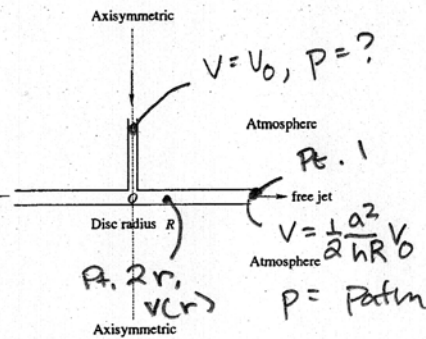
Pr. 3 stats  
 Mean: 26.8/70  
 St Dev: 17.8

70  
 3. (85) The figure shows the side view of two discs each of radius  $R$ , and separated by fixed distance  $h$ . Air enters the gap between the discs with speed  $V_0$  through the supply tube of radius  $a$ , flows radially and leaves as a free jet into the atmosphere. Atmospheric pressure is  $p_a$ . Within the gap, the speed  $V$  varies with distance  $r$  from the axis according to the rule

$$V = \begin{cases} \frac{1}{2} \frac{r}{h} V_0 & \text{if } r < a \\ \frac{1}{2} \frac{a^2}{hr} V_0 & \text{if } r > a. \end{cases}$$

(You are not required to show that.) The pressure within the supply tube is unknown; it is not atmospheric.

\* If you used Bernoulli w/ one pt @ supply WITH  $p = p_{atm}$ , you got a max of +5 for (a)



(40 max) (a) Using the Bernoulli equation, find the pressure  $p$  within the gap as a function of  $r$ ,  $\rho$ ,  $V_0$ ,  $h$  and  $a$ . Sketch  $p$  as a function of  $r$  showing clearly any maxima, minima and zeros. (You need to consider the cases  $r < a$ ,  $r > a$  separately.)

BE along SL from  $r$  to exit Pt 2

$$\Rightarrow \frac{1}{2} \rho V^2(r) + p(r) = \frac{1}{2} \rho V_e^2 + p_a$$

$$\Rightarrow p(r) - p_a = \frac{1}{2} \rho V_e^2 \left\{ 1 - \frac{V^2(r)}{V_e^2} \right\}$$

+20 for properly using Bernoulli properly. This includes those who tried to find  $p_{supply}$

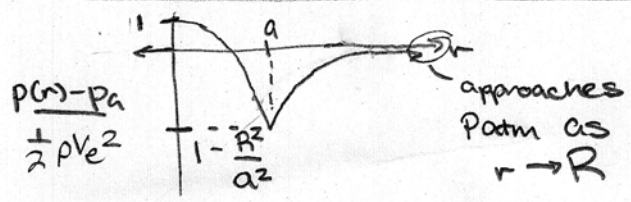
But  $V_e = \frac{1}{2} \frac{a^2}{hR} V_0 \Rightarrow \frac{V}{V_e} = \begin{cases} \frac{Rr}{a^2} & r < a \\ \frac{R}{r} & r > a \end{cases}$

$$\Rightarrow \frac{p(r) - p_a}{\frac{1}{2} \rho V_e^2} = \begin{cases} 1 - \frac{R^2}{a^2} r^2 + 5, & r < a \\ 1 - \frac{R^2}{r^2} + 5, & r > a \end{cases}$$

1s08-3

Boxed EQNS GIVE THE SOLN

since  $R > a$   
 $\frac{R}{a} > 1$



+5 for correct plot  
 +5 for correct labeling + comments

(20/70) (b) Noting that the flow is axisymmetric, derive the integral expressing the force  $F$  exerted by the air on both sides of the lower disc in terms the pressure difference  $p(r) - p_a$ .

Area element is a ring of radius  $r$ , thickness  $dr$ .

REQUIRED FOR FULL CREDIT  $\rightarrow$   $dA = 2\pi r dr$  or  $dA = r dr d\theta$

Vertical force on  $dA = 2\pi (p_a - p(r)) r dr$

Force upward on lower disc =  $2\pi \int_0^R (p_a - p(r)) r dr$

PLAN VIEW

(c) Hence show that  $F$  is given by the expression

$$F = \frac{1}{4} \pi \rho \frac{a^4}{h^2} V_0^2 \left( \ln \frac{R}{a} - \frac{1}{4} \right).$$

Explain physically why, despite the presence of a stagnation point at  $O$ , the force is upwards if  $R \gg a$ .

$$\begin{aligned} \frac{F}{\rho \pi V_0^2} &= - \int_0^R \left( 1 - \frac{v^2}{V_0^2} \right) r dr \\ &= - \int_0^a \left( 1 - \frac{R^2}{a^4} r^2 \right) r dr - \int_a^R \left( 1 - \frac{R^2}{r^2} \right) r dr \\ &= - \left[ \frac{1}{2} r^2 - \frac{R^2}{4a^4} r^3 \right]_0^a - \left[ \frac{1}{2} r^2 - R^2 \ln r \right]_a^R \\ &= R^2 \ln \frac{R}{a} - \frac{1}{4} R^2 \end{aligned}$$

Subst for  $V_0$

i.e.  $\frac{F}{\frac{1}{4} \pi \rho \frac{a^4}{h^2} V_0^2} = R^2 \left( \ln \frac{R}{a} - \frac{1}{4} \right)$

$$\Rightarrow \frac{F}{\frac{1}{4} \pi \rho \frac{a^4}{h^2} V_0^2} = \ln \frac{R}{a} - \frac{1}{4}$$

END

+5

- Requires full + correct integration of correct  $v$  function

+5

1s08-4

Although near stagnation  $p(r) > p_a$ , over most of the disc is  $p(r) < p_a$ . Consider mass conservation. For  $r > a$ , as  $r$  inc,  $v$  dec.  $\therefore p$  increases, but must reach  $p_a$  at jet.  $\therefore p(r) < p_a$  for  $r > a$