

Mean: 144/200
 Standard Deviation: 44

1. (65) In a manufacturing process, incompressible viscous polymer is squeezed between a stationary die, and descending mold. As a result, polymer is squeezed axisymmetrically from the gap; all 3 Cartesian components v_x , v_y and v_z of the velocity vector are non-zero. The x and y velocity components are given by the expressions

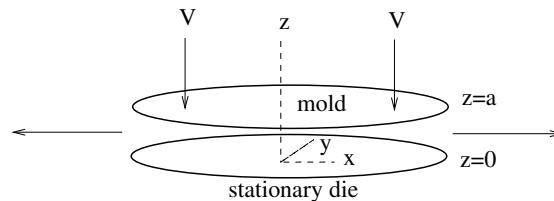
$$v_x = kx(az - z^2), \quad v_y = ky(az - z^2).$$

Mean: 50/65
 Standard Deviation: 18

Using the continuity equation, namely

$$\frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z} = 0$$

and the boundary conditions on v_z , derive the expression giving v_z as a function of z , a and the (positive) speed V . (The parameter k must **not** enter into your final answer.)



SOLUTION.

Substituting v_x , v_y into the continuity equation, we obtain

$$2k(az - z^2) + \frac{\partial v_z}{\partial z} = 0. \quad \text{+ 25 points}$$

Integrating from $z = 0$ to z , and applying the b.c. $v_z = 0$ on $z = 0$, we find that

$$-v_z = \frac{1}{3}k(3az^2 - 2z^3) \quad \text{+ 10 points} \quad (1)$$

Applying the remaining b.c. $v_z = -V$ on $z = a$, and solving for k , we obtain

$$k = 3V/a^3. \quad \text{+ 10 points}$$

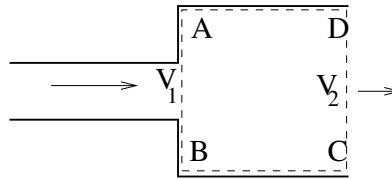
Substituting for k into Eq.(1), and rearranging, we find that

$$v_z = -V \left\{ 3 \left(\frac{z}{a} \right)^2 - 2 \left(\frac{z}{a} \right)^3 \right\}. \quad (QED)$$

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2. (65) Mass conservation requires an incompressible fluid to decelerate as it flows through a sudden expansion in a duct, so that the pressure rises from AB to CD. Assuming that everywhere on AB, $p = p_1$, and that on CD, $p = p_2$, where $p_1 - p_2 = \rho V_2(V_2 - V_1)$, and using the control volume shown, derive the expression giving the mechanical energy loss ΔE per unit mass flowing through the expansion in terms of V_1 and V_2 . **Your answer must show explicitly that $\Delta E \geq 0$.**



Mean: 53/65
Standard Deviation: 16

SOLUTION.

Balancing mechanical energy, we find that

$$\dot{m} \left[\frac{1}{2} V^2 + \frac{p}{\rho} \right]_1 = -\dot{m} \Delta E.$$

+ 30 points
(-5 points if starting from Balance of Total Energy)

(Terms representing shaft power and change in gravitational potential energy vanish identically.) Cancelling \dot{m} , and substituting for $p_1 - p_2$, we obtain

$$\frac{1}{2}(V_2^2 - V_1^2) - V_2(V_2 - V_1) = -\Delta E.$$

+ 15 points

Rearranging, we obtain the following:

$$\Delta E = \frac{1}{2}(V_1 - V_2)^2. \quad (QED)$$

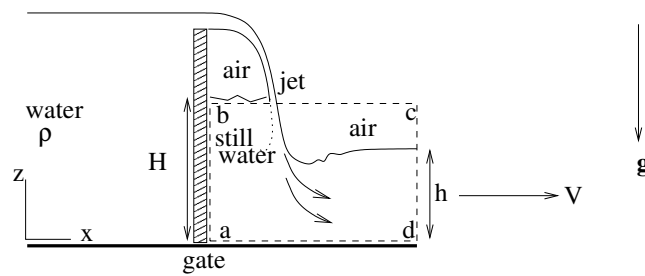
+ 20 points

The mechanical energy loss ΔE per unit mass is positive, as required by the second law of thermodynamics.

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3. (70) Water flowing over a tall sluice gate in a river re-enters the river downstream almost vertically. Because there is a net flow of horizontal momentum out of the control volume illustrated, a horizontal force must act on the material within the control volume. As a result, water backs up to depth H between the gate and the vertical jet; the horizontal force results because $H > h$. Assuming that the water of depth H is stagnant, and that the flow downstream is purely horizontal with speed V , derive the equation giving H in terms of h , V and g .



Mean: 42/70
Standard Deviation: 24

SOLUTION.

The net flow rate of x -momentum out of control volume $abcd$ is $(\rho V)(Vh)$ (per unit length into the page).

+ 20 points

The only forces exerted in the x -direction on matter in the control volume are the pressure forces exerted on surfaces ab and cd .

On surface ab , the pressure at height z above the river bottom is given by $p = p_A + \rho g(H - z)$. The x -component of force acting on a strip dz is $p dz$, and the resultant horizontal force is given by

$$F_{ab} = \int_0^H p dz, = p_A H + \frac{1}{2} \rho g H^2.$$

+ 10 points

On surface cd , the pressure force acts to the left, and is given by $F_{cd} = - \int_0^H p dz$. In calculating the integral, we note that in the air for $z > h$, $p = p_A$; in the water for $z < h$, $p = p_A + \rho g(h - z)$. Evaluating the integral, we find that

$$F_{cd} = -p_A H - \frac{1}{2} \rho g h^2.$$

+ 10 points

The resultant rightward force is $F_{ab} + F_{cd} = \frac{1}{2} \rho g (H^2 - h^2)$. We note that this is independent of p_A ; equally, we could have used the gauge pressure.

The balance of x -momentum is as follows:

+ 30 points

$$\rho V^2 h = \frac{1}{2} \rho g (H^2 - h^2).$$

Solving for H we find that

$$H = h \sqrt{1 + 2 \frac{V^2}{gh}} \quad (QED)$$

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